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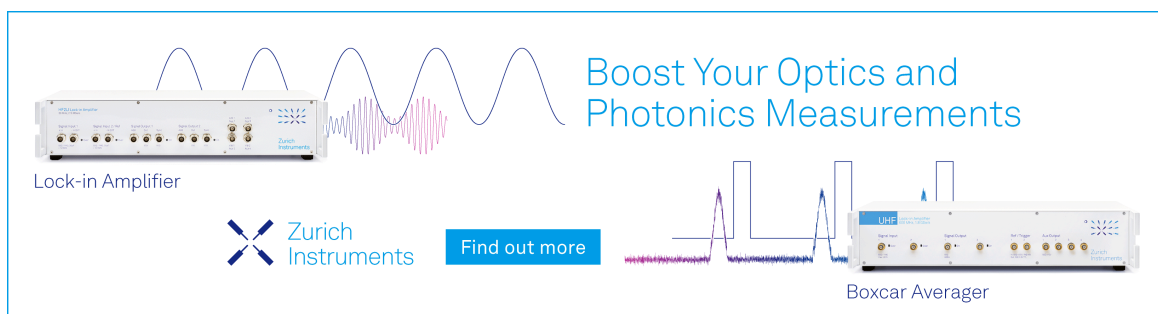
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Mikhail Konstantinov , Ivan Gluchkov; Igor Gerasimenko; Igor Ognev; Alexey Gretsov

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Justification of the Parameters and Operating Modes of the Fan of the Device for Collecting Grain, which is Part of the Reaper Designed for Two-Phase Harvesting in a Batch Way

Mikhail Konstantinov^{1, a)}, Ivan Gluchkov^{1, b)}, Igor Gerasimenko¹, Igor Ognev^{2, c)},
and Alexey Gretsov³

¹*Orenburg state agrarian University, Orenburg, Russia*

²*Ural Federal University named after the first President of Russia B.N. Yeltsin, Ekaterinburg, Russia*

³*Samara State Agrarian University, Samara, Russia*

a) Corresponding author: miconsta@yandex.ru

b) i-n-g2012@yandex.ru

c) ognev.i.i@yandex.ru

Abstract. Studies of the reaper designed for two-phase harvesting by batch method, conducted earlier showed the prospects for the development and research of technical means to preserve grain that can already be considered lost. As a solution to the problem, a modernized reaper designed for two-phase harvesting by batch method was developed, in the design of which a device was introduced for collecting grain that came out of the ear in contact with the main nodes of the header, which was called a device for collecting free grain. The fan is an important element of the device for collecting grain, so the justification of its parameters and operating modes described in this article is relevant. In the course of solving this problem, mathematical expressions were obtained that allow us to justify the choice of a fan for the particular case under consideration – for a device for collecting free grain that complements the design of a batch header. The article explains the choice of a specific type of fan. As conclusions on the article under consideration, the results of the conducted research are shown. In particular, the ranges of optimal values of a number of parameters of the considered fan, such as, for example, the power consumption of the fan and its performance, are presented. All the research results shown in the article relate directly to a particular case of a fan – a fan that is part of a device for collecting free grain.

INTRODUCTION

The conducted studies of the reaper designed for two-phase harvesting by batch method, its design and technological features, allowed us to identify the optimal operating parameters of its operation and justify its design features in such a way as to minimize the loss of grain of all types [1, 2]. However, it should be understood that it is impossible to reduce grain losses to zero due to many factors – the inability to constantly maintain an optimal combination of natural and climatic and time intervals of harvesting, the heterogeneity of agrobiological properties and qualities of the harvested crop even on the scale of a single field, a significant gradation of weather conditions, the presence of vibrations and inertial processes inevitable for any mechanism, etc. [3-5]. This conclusion showed the prospects of research on reducing grain losses in another direction – the development of technical means to save grain, which in the context of standard technology can already be considered lost [6]. In addition to directly developing such a device, the justification of its technical and technological parameters is of indisputable importance.

METHODOLOGY

As a solution to the above problem, a modernized reaper designed for two-phase harvesting by batch method was developed, the design of which was introduced with a device for collecting free grain (RF patent No. 2523847), i.e. grain material that came out of the ear in contact with the main nodes of the reaper or due to over-ripening of the harvested crop in the field (figure 1).

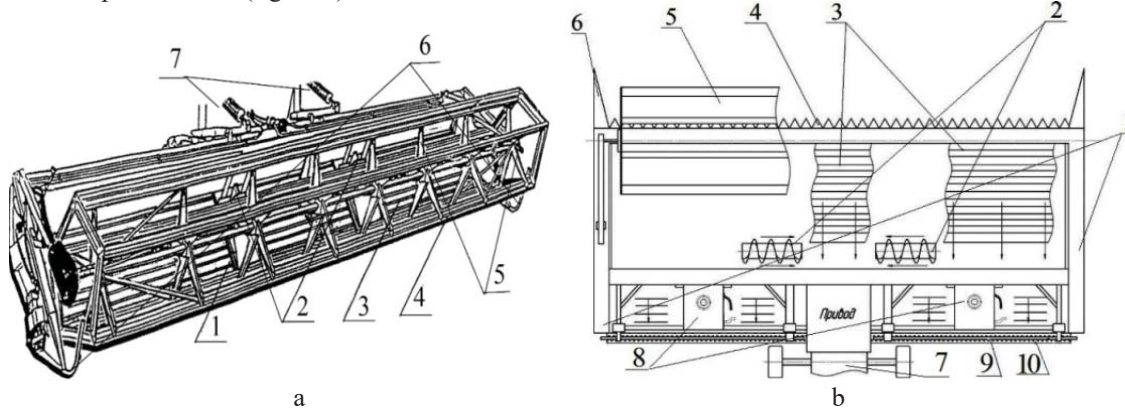


FIGURE 1. Design scheme of a reaper designed for two-phase harvesting by batch method with a device for collecting free grain: a – general view, b – top view: 1 – reaper frame; 2 – devices to prevent cut stems from getting under the wheels; 3 – conveyor belt; 4 – the cutting apparatus; 5 – the device for bringing the stalks to the cutting apparatus; 6 – dividers; 7 – reaper hitch; 8 – hopper of the device for free grain; 9 – sieve with funnels of the device for free grain; 10 – discharge gate

In the design and technological scheme of the device for collecting free grain, the fan is one of the main components. In this regard, both from a scientific and practical point of view, it is important to identify and justify the patterns that determine its operating modes and parameters, as well as the factors that influence them. Justification of these indicators and identification of their optimal values will optimize the process of preventing grain losses during the operation of the conveyor.

A radial fan is selected as the fan of the device for collecting free grain. The choice in favor of this type of fan is explained by the fact that it is not characterized by the well-known disadvantage of axial-type fans, which consists in the fact that the impact of the fan blades on the air particles is not the same along their length, that is, the closer the air particles are located to the center, the worse they are captured by the blades. In addition, axial fans create a relatively small air vacuum [7, 8].

The choice of the fan and its operating modes was justified taking into account its main parameters, such as drive power, fan speed, as well as the required air consumption [7, 9].

The optimal power for the fan drive can be set as follows [7]:

$$N_e \geq \frac{K_{zy} \cdot V \cdot P_e}{\eta_{np} \cdot \eta_e}, \text{ W} \quad (1)$$

where K_{zy} - stock ratio, equal to 1.5; V - required air flow, m^3/s ; P_e - pressure in the device for collecting free grain, Pa; η_{np} - efficiency factor of the drive; η_e - efficiency factor of the fan.

$$P_e = 10^5 - \Delta P_\Sigma, \text{ Pa} \quad (2)$$

where ΔP_Σ - total pressure loss:

$$\Delta P_\Sigma = \Delta P_B + \Delta P_M + \Delta P_{II} + \Delta P_p, \text{ Pa} \quad (3)$$

where ΔP_B - pressure loss in the pipeline during the movement of clean air, Pa; ΔP_M - pressure loss during material movement, Pa; ΔP_{II} - pressure loss when lifting the material, Pa; ΔP_p - acceleration pressure loss, Pa [10].

$$\Delta P_B = \lambda_e \frac{L_{np,e}}{d_T} \cdot v_e^2 \cdot \gamma_e, \text{ Pa} \quad (4)$$

where λ_g - coefficient of resistance when moving clean air; $L_{np.g}$ - the reduced length of the transport pipeline when moving clean air, m; d_T - pipeline diameter, m; v_g - speed of the conveying flow, m/s; γ_a - air density, kg/m³.

When clean air is moving, the reduced length of the transport pipeline is equal to [7]:

$$L_{np.g} = L_z + L_g, \text{ m} \quad (5)$$

where L_g - length of the vertical part of the pipeline, m; L_z - length of the horizontal part of the pipeline, m;

The value of the diameter of the device pipeline is defined as:

$$d_T = \sqrt{\frac{Q}{\pi \cdot v_g \cdot \mu}}, \text{ m} \quad (6)$$

where Q - productivity, kg/s; μ - weight concentration of the mixture, kg/m³.

In the case of working with a complex mixture – in our case, it is a mixture that includes grain, air and possible impurities – the value of the weight concentration will be 25 kg/m³ [11].

The required productivity of the device for collecting free grain is defined as:

$$Q = \frac{B \cdot \Delta \cdot \Psi \cdot b_{m.g.} \cdot b_{\text{жс}}}{t_{pa3}}, \text{ kg/s} \quad (7)$$

where B - crop yield on the field in terms of per square meter, kg/m²; Δ - coefficient of grain loss before contact with the device; Ψ - coefficient that takes into account the compensation of losses while maintaining free grain, 0.65...0.90; $b_{m.g.}$ - roll spacing (correction for areas of the header that are not occupied by the width of the conveyor), m; $b_{\text{жс}}$ - conveyor width, 8m; t_{pa3} - time of unloading a portion of bread mass, c.

The unloading time is calculated as follows:

$$t_{pa3} = \frac{b_g}{v_{arp}}, \text{ c} \quad (8)$$

where b_g - roll width, m; v_{arp} - reaper speed, m/s.

Substituting the expressions (7) and (8) into the dependence (6), and also taking into account that the values of some components in these expressions are known and constant, we obtain the following regularity for determining the diameter of the pipeline, which is characteristic of our particular case:

$$d_T = \sqrt{\frac{0,32 \cdot B \cdot \Delta \cdot \Psi \cdot b_{m.g.} \cdot v_{arp}}{\pi \cdot v_g \cdot b_g}}, \text{ m}, \quad (9)$$

where 0,32 – a constant obtained as the ratio of the known values of the header width to the weight concentration of the mixture.

According to the results of the performed studies, the optimal values of the pipeline diameter of the free grain collection device were identified, which are in the range of 0.16-0.22 m.

During the movement of the material, the pressure loss will be determined as follows:

$$\Delta P_M = \lambda_M \cdot \mu \frac{L_{np.M}}{d_T} \cdot v_g^2 \cdot \gamma_a, \text{ Pa} \quad (10)$$

where λ_M - the drag force during the movement of the mixture and air, which is selected according to the schedule; $L_{np.M}$ - the reduced length of the pipeline for the case of movement of clean air with the material, which is set by the formula:

$$L_{np,m} = L_c + L_e + L_{\text{лсМ}}, \text{ m} \quad (11)$$

where $L_{\text{лсМ}}$ - pipeline length for local equivalent resistance, m.

To determine the pressure loss on the lifting of the material, you can do the following:

$$\Delta P_n = \mu \cdot \gamma_e \cdot H \cdot g \cdot \frac{v_e}{v_m}, \text{ Pa} \quad (12)$$

where H - the height of the vertical section of the pipeline, m; v_m - material movement speed.

Next, we determine the value of the pressure loss on the acceleration of the material:

$$\Delta P_p = \Omega \cdot \varepsilon \cdot \mu \cdot v_e^2 \cdot \gamma_e, \text{ Pa} \quad (13)$$

where Ω - number of inputs; ε - bending coefficient, (determined from the catalog, depends on the radius and angle of bending and the Reynolds number).

Can be can set the required air flow V for the current performance by using the following expression:

$$V = \frac{\pi \cdot d_T^2}{4} v_e, \text{ m}^3/\text{s} \quad (14)$$

Substituting the expressions (2) and (14) in the formula (1), we get:

$$N_e \geq \frac{K_{\text{зв}}}{\eta_{np} \cdot \eta_e} \cdot \frac{\pi \cdot d_T^2}{4} v_e \cdot \left(10^5 - \left[\frac{\lambda_e \cdot L_{np,e} + \lambda_m \cdot \mu \cdot L_{np,m}}{d_T} + \frac{\mu \cdot H \cdot g}{v_e \cdot v_m} + \Omega \cdot \varepsilon \cdot \mu \right] \cdot v_e^2 \cdot \gamma_e \right) \quad (15)$$

RESULTS AND DISCUSSION

Performing some transformations of the resulting expression, taking into account the features characteristic of our particular case, took into account, among other things, that for grain, the speed of the transporting flow is 22 m/s [11], and the air density is selected depending on the design of the device according to the catalog – based on the operating conditions and features of the device known to us, we choose the air density value equal to 12 kg/m³ [7]. Accordingly, the components v_e^2 and γ_e for our case have constant and known values, and their product is 5808. You should also pay attention to the fact that the stock ratio of $K_{\text{зв}}$ is 1.5:

$$N_e \geq \frac{0,12 \cdot B \cdot \Delta \cdot \psi \cdot \varepsilon_{\text{м.е.}} \cdot v_{\text{азр}}}{\eta_{np} \cdot \eta_e} \cdot \left(10^5 - 5808 \cdot \left[\frac{\lambda_e \cdot L_{np,e} + \lambda_m \cdot \mu \cdot L_{np,m}}{d_T} + \frac{\mu \cdot H \cdot g}{v_e \cdot v_m} + \Omega \cdot \varepsilon \cdot \mu \right] \right) \quad (16)$$

In order to bring the above regularity to a more compact and convenient expression, we will replace some of the components that are inherently close to constant values with symbols:

$$K_1 = \lambda_e \cdot L_{np,e} + \lambda_m \cdot \mu \cdot L_{np,m}, \text{ m}; \quad K_2 = \frac{\mu \cdot H \cdot g}{v_e \cdot v_m}, \text{ m/c}; \quad K_3 = \Omega \cdot \varepsilon \cdot \mu$$

Taking into account the above, we will write down the final form of the resulting formula:

$$N_e \geq \frac{0,12 \cdot B \cdot \Delta \cdot \psi \cdot \varepsilon_{\text{м.е.}} \cdot v_{\text{азр}}}{\eta_{np} \cdot \eta_e} \cdot \left(10^5 - 5808 \cdot \left[\frac{K_1}{d_T} + K_2 + K_3 \right] \right), \text{ W} \quad (17)$$

CONCLUSIONS

The conducted studies allowed us to justify the ranges of optimal values of a number of parameters of the fan of the device for collecting free grain. In particular, the fan power consumption of the device is in the range of 1.03-1.15 kW, the performance when operating without grain (determined for correlation with the factory data of serial fans when evaluating their suitability for use in the device in question) – 0.412-0.476 m³/s, the rotation speed-up to 43c⁻¹, and the created pressure (discharge) in the system-from 2.8 to 3.2 kPa.

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